

Wheel squeal and flanging noise Challenges in validation of friction management in the field and laboratory

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30th June 2020

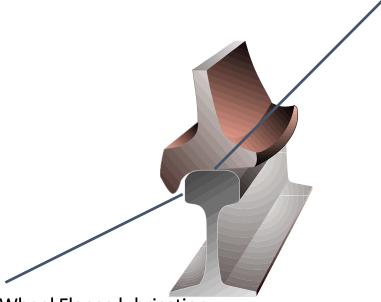


Contents

- > Background to rail-wheel noise
- > Wheel squeal field trials
- > Theory behind wheel squeal
- > Laboratory testing
- > Summary of challenges



Friction Management – Guiding Principles



Sauge Face (GF) / Wheel Flange lubrication

TARGET: COF < 0.15

- > Impacts:
 - Rail / Wheel Wear
 - Gauge Corner Cracking
 - Flange Noise
 - Derailment Potential (Wheel Climb)

Top of Rail (TOR) / Wheel Tread Friction Modifier

TARGET: COF ~ 0.35

- > Typical dry 0.6
- > Impacts:
 - Rail / Wheel Wear
 - RCF Development
 - Top of Rail Noise
 - Corrugations
- > Reduced lateral forces
- > Switch protection
- > Reduced traction energy consumption



Background to rail wheel noise

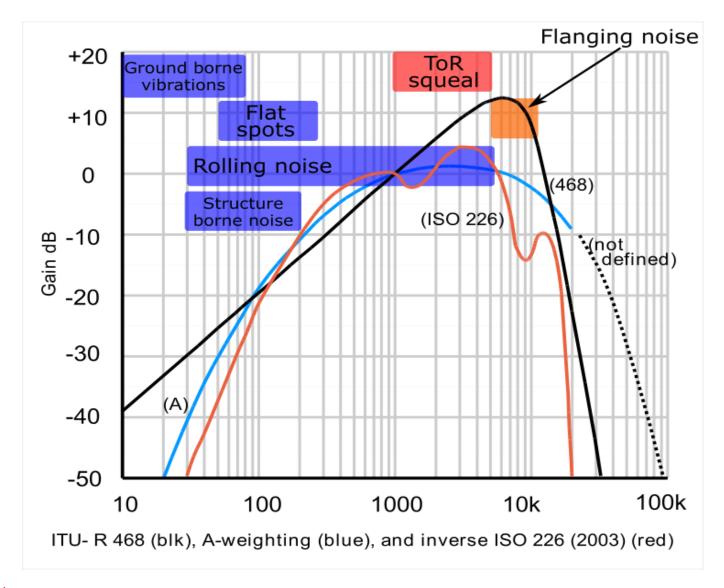


Noise: Spectral ranges

Noise Type	Frequency range [Hz]		
Rolling	30 – 2500		
Rumble (including corrugations)	200 – 1000		
Flat spots	50 – 250 (speed dependant)		
Ground Borne Vibrations	30 – 200		
Top of rail squeal	1000 – 5000		
Flanging noise	5000 — 10000		



Human perception of noise





Squeal and Flanging Noise

Top of rail wheel squeal noise



- High pitched, tonal squeal (predominantly 1000 5000 Hz)
- Prevalent noise mechanism in "problem" curves, usually < 300m radius
- Related to both negative friction characteristics of Third Body at tread / top of rail interface and absolute friction level
 - Stick-slip oscillations
 - Leading wheelset, inside wheel

Flanging noise



- Typically a "buzzing" OR "hissing" sound, characterized by broadband high frequency components (>5000 Hz)
- Affected by:
 - Lateral forces: related to friction on the top of the low rail
 - Flanging forces: related to friction on top of low and high rails
 - Friction at the flange / gauge face interface



Corrugation noise



Noise due to corrugation with occasional wheel squeal and flanging noise

Corrugation noise

 Low pitched rumbling noise due to the presence of corrugation on the running band of the rail



Field trials



FM Focus: Noise/Corrugation

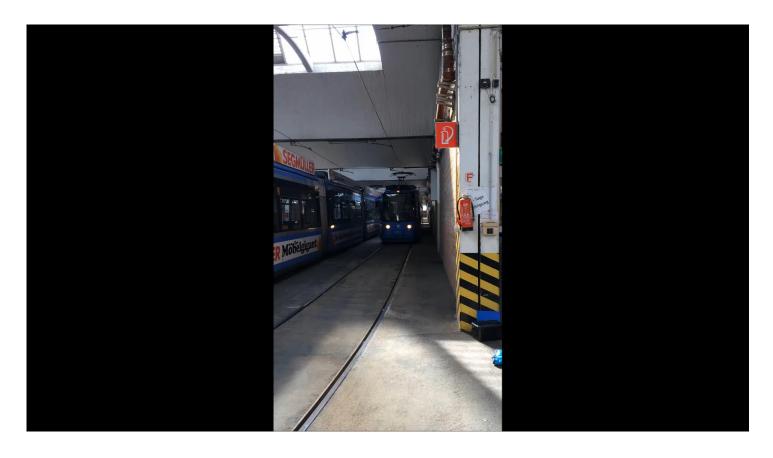
Baseline – **No TOR FM** application





FM Focus: Noise/Corrugation

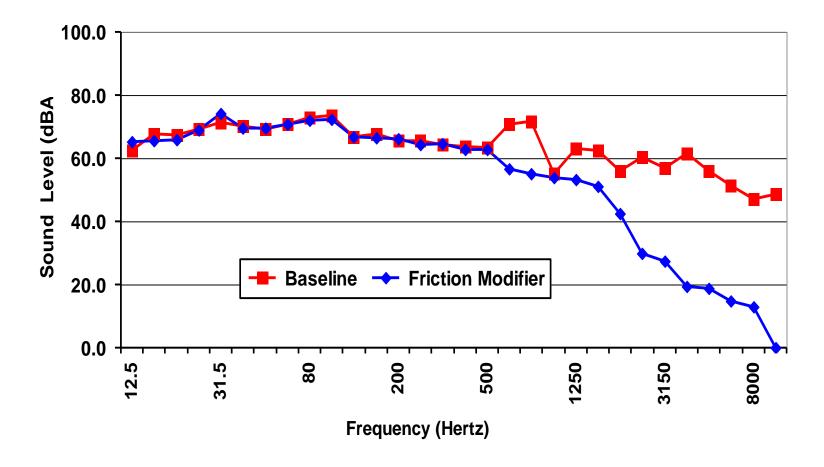
AFTER TOR FM application - manual





Spectral sound distribution: Trams

> Effects of frictional conditions

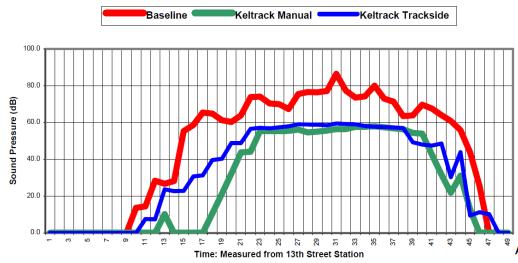




Spectral sound distribution: Trams

> Fffects of frictional conditions

Average Sound Pressure versus time at 1.6 kHz



Average Sound Pressure vs. Time at 2.5 KHz





Field trials

- > Typical field trials compare baseline measurement to application of top of rail materials
- > Noise can be very specific to:
 - Vehicle
 - Bogie steering primary yaw stiffness
 - Wheel profile
 - Location curve/cant
 - Running speed
 - Weather
 - Rain and moisture (morning dew) particularly large impacts
 - Humidity
 - Rail and wheel contamination
 - Measurement location (reflections)
- > Don't necessarily get squeal from every bogie

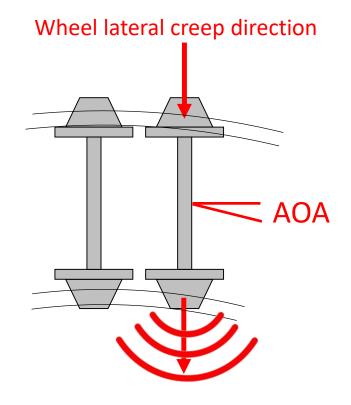


Wheel squeal – conventional theory

Lateral creepage of the wheel - prime cause of squeal

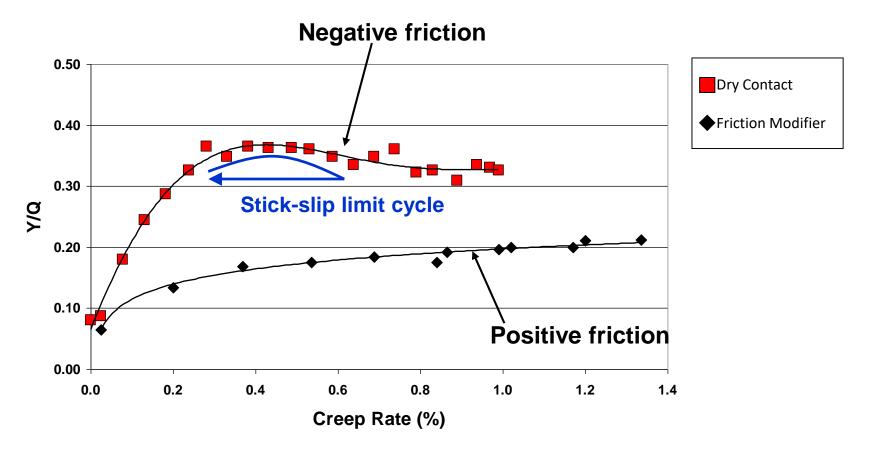
- Particularly for the leading inner wheel of a bogie
- stick-slip mechanism of this creep force

Rudd 1976, Remington 1985





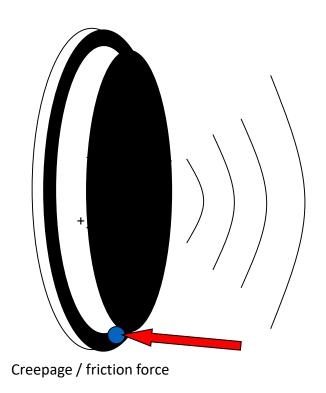
Absolute Friction Levels and Positive/Negative Friction – conventional theory



^{*} Replotted from: "Matsumoto a, Sato Y, Ono H, Wang Y, Yamamoto Y, Tanimoto M & Oka Y, Creep force characteristics between rail and wheel on scaled model, *Wear*, Vol 253, Issues 1-2, July 2002, pp 199-203.



Absolute Friction Levels and Positive/Negative Friction – conventional theory



Frequency response of the wheel

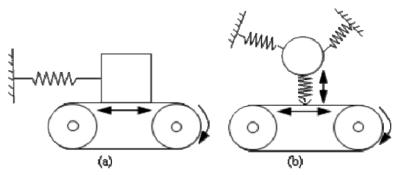


Mode Coupling Instability Mechanism

- > Jiang, Anderson and Dwight, 2015
- > Further analysis by Bo Ding 2018

Theory:

- > Based on commonly accepted theory for squeal in braking systems. A coupling of vibration in two different directions
- > Wheel/rail interface subject to vertical and lateral vibrations and forces





Mode Coupling Instability Mechanism

> The lateral frictional force between the wheel and the rail is related to the normal (vertical) force, so a natural coupling

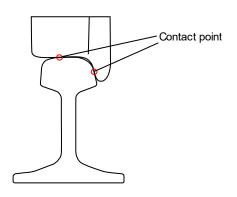
- $> F = \mu N$
- > If wheel vertical and lateral vibration frequency modes are close
- > Then friction coefficient increases to a critical threshold,
 - -> vertical and lateral oscillations become out-of-phase
 - -> friction force, which depends on the vertical force, is therefore out-of-phase with the lateral motion => unstable positive feedback.



- >Mode coupling instability mechanism
- > Curley, Anderson, Jiang and Hanson track study
 - Found noise from wheels on inner and outer rail
 - Running bands in different locations on gauge corner of inner and outer rail
 - Track form had an influence
 - Found TOR FM had benefit when applied to both rails, but in some cases when applied to inner rail only no benefit
 - Counter to normal theory gauge corner lubrication also had a benefit for wheel squeal



- >Mode coupling instability mechanism
- > For all these theories friction between the wheel and the rail still key.
- > Work carried out by Bo Ding 2018 studied slip/stick mechanism, mode coupling instability, and third potential mechanism wheel rail coupling
- > 2 point contact not studied much wrt wheel squeal

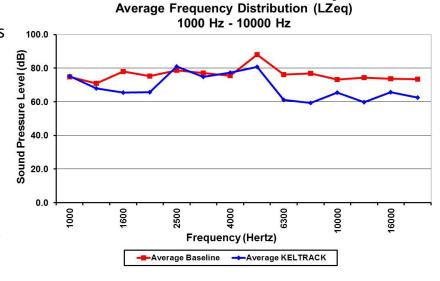




Freight trial with two types of water based TOR FMs



- > Two versions of top of rail friction modifiers tested
 - Both products have high positive friction characteristics
 - Similar intermediate friction
 - Different binders and tackiness
 - One product retained more on wheel (less effective), other transferred more to rail
 - One product much more effective than the other in noise reduction
- > Oil and grease based top of rail materials difficult to balance noise reduction with braking and traction performance
- > Difficult to predict noise performance from laboratory testing





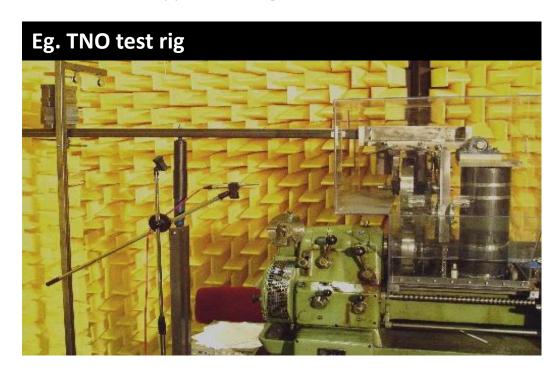
Laboratory investigations into wheel squeal

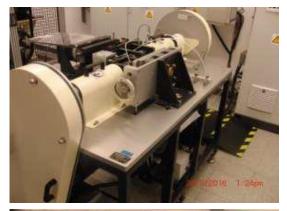


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Laboratory testing

> Twin disc type testing



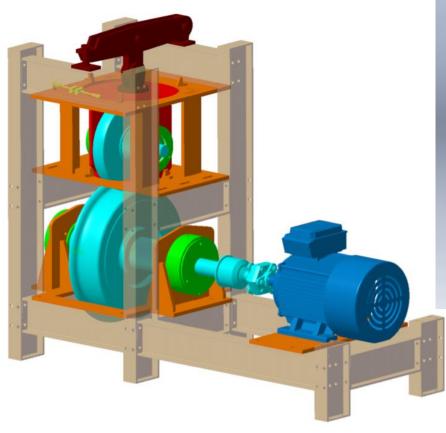


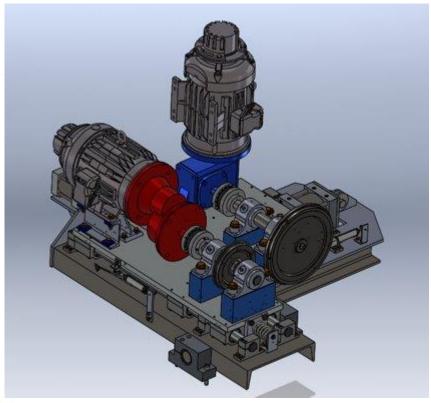




Laboratory testing

> Scaled rigs

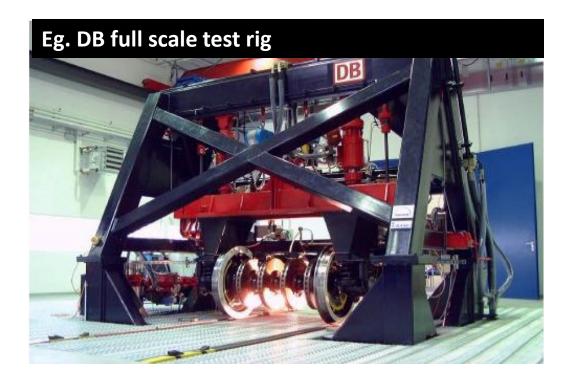






Laboratory testing

> Full scale test rigs





UIC study 2005

- > Compared results wheel squeal mitigation of different products on laboratory test rigs and on site measurements
 - Issue of application rate, too

		TOR FM1	TOR FM2	TOR FM3	TOR FM4	
		(water	(oil	(oil	(oil	
		based)	based)	based)	based)	Water
Lab	TNO rig	Υ	Υ	Υ	Υ	
	DB rig	Υ	Υ			Υ
Field	Site 1	Υ	N	N	N	
	Site 2	N				
	Site 3	Υ				Υ

Comparison of wheel squeal for different FMs in lab and in the field – (Y – Noise reduced, N – no significant noise reduction)



Wheel squeal mitigation

- > Good understanding of effective mitigation methods
 - Top of rail/tread friction modifiers (lubrication) all theories point to the importance of friction control
 - Bogie/wagon design
 - Distance between axles
 - Vehicle steering primary yaw stiffness
 - Wheel dampeners
 - Wheel and rail profile
 - Track form dynamics



Challenges



Product development

- Need on track trials and case studies to prove noise reducing properties.
- > Lab scale test can give an indication but not the whole story
 - Slows down product development need to produce larger batches
 - Some products are easier to test by manual application than others
 - Eg on board solid stick friction modifiers and lubricator sticks

 to test in the field do you swap out the sticks from the
 whole fleet to negate the effect of other sticks on the
 performance? In a smaller limited trial do you build up
 sufficient film thickness
- > Need for better lab scale noise testing
- > Understanding of required film thickness/application rate



End customer

- > Needs case studies and evidence of on track performance
- > Cannot rely on lab tests and friction data alone
- > Squeal/noise remains a major issue for most railways/metros

"curve squeal remains one of the least understood railway noise sources despite the continuing efforts over recent decades" Jiang, Anderson and Dwight, 2015



Questions